

Experimental Tests of Transonic Nozzles in Linear Cascades Carried out by Traversing Probe

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Abstract

The purpose of this paper is to present the experimental technique developed at D.I.ME.CA. (Department of Mechanical Engineering of Cagliari) for aerodynamic studies in high speed linear cascades. Because of the limited running time of the wind tunnel (blow-down type) the experiments have been done with a limited number of blades and minimizing the acquisition time of the various measurements consistently with the time relative to the traversing probe. The measurements obtained have shown a good accuracy and the results can be considered more than satisfactory in characterizing blade performance.

Nomenclature.

ξ_2 = loss coefficient = $1 - w_2^2/w_{2,is}^2$

THL2 = total head loss =

$= (1 - (P_{02}/P_{01})^{(k-1)/k}) / (1 - (P_{s2}/P_{01})^{(k-1)/k})$

P_0 = total pressure

P_s = static pressure

Subscripts.

1 = plane upstream

2 = traversing plane downstream.

1. Introduction

It is well understood that the availability of experimental data is of fundamental importance to the improvement of the performance of turbomachines.

It permits us to compare different solutions adopted and therefore allows us to continually refine technical design and validate test data with mathematical models.

Today, the accuracy required for experimental analysis is continually increasing and this represents a particular difficulty for the researcher, especially if the research is on cascades working in transonic and supersonic conditions.

Even if many fluid dynamic laboratories are directing their attention towards rotating annular cascade test rigs, experimental studies in linear cascades remains valid particularly in high speed turbine blades, where they will continue to make a notable contribution.

On the other hand, contemporary progress in measuring instrumentation allows us to experiment cascade flows in reduced time, making it possible to achieve good results whilst employing small wind tunnels and therefore reducing costs.

2. Test Rig.

Starting from this consideration, within D.I.ME.CA., the high speed wind tunnel, which for some time has been utilized for aerodynamic probe research [1], [2], [4], [5], [6], has been modified to allow experimental research on linear cascades

[5], [6], has been modified to allow experimental research on linear cascades operating in transonic and supersonic conditions.

The wind tunnel is a blow-down type, which exhausts to atmosphere, incorporating a rectangular test section with a variable height from 70 to 210mm and with a width of 70mm.

The storage volume is 25m^3 at a pressure of 1.6Mpa, with an installed power of 100kw.

At time, it is possible to run continuously for periods from 40 to 120 seconds, with a variable number of blades. (from 6 to 4), depending on the dimensions and the type.

Because of the limited running time, it has been necessary to develop a measuring system with relative reduced response time to allow the acquisition of all measurements required to evaluate the cascade performance.

The acquisition system is composed of an HP 9000 Model 360 controller, interfaced by an IEEE 488, to an HP 6942 Multiprogrammer.

The wall static pressures, upstream and downstream, and the blade pressures are measured by means of two X 48 channel scanivalves each with a pressure transducer incorporated, while the aerodynamic probe pressures are measured with separate transducers.

All of the transducers used are of the same type (Scanco PDCR 24) with a response time less than 20ms with a maximum tube length, between probe head and transducer, of 70cm.

For the measurement of the probe position, during test running, an high accuracy linear potentiometer is used (Leane HPL 190).

All of the measurements are taken 8 times, to improve accuracy, with a frequency of 25kHz.

In this way the total measurement time depends on the scanning time of the scanivalves (more than 50ms), but, above all, on the time traversing probe.

The probe travels at a constant speed but the reading of pressure is limited to 100 step positions with a step distance (Ds) equal to or greater than 1mm, and a forward velocity (Vp) variable from 1 to 2.5mm/s, chosen in such a way that the displacement of the probe can be considered negligible in comparison to the step distance ($D_s < 0.005\text{mm}$ for $V_p=2.5\text{mm/s}$).

Flow control downstream of the cascade is achieved by means of a suitably perforated tailboard.

For the preliminary control of the flow periodicity the Schlieren system is used in conjunction with the wall downstream static pressures and the trailing edge pressures on the instrumented blades.

The aerodynamic probe used in the traverse is a cone-probe type, [4] [16], shown in Fig 1. and during test it is positioned at an axial distance equal to 1/2 of one pitch.

Usually the traverse is equal to 3 pitches which also permits verification of the flow periodicity with the pressure readings.

The results presented in the paper refer to a steam turbine nozzle cascade (shown in Fig 2) where the outlet velocity is supersonic.

3. Measurements Analysis

The individual tests have each been repeated several times, varying the scanivalve scanning time and the speed traversing probe, to verify the measurements accuracy. The observed differences show variations of the order of .003 for the loss coefficient, ξ_2 , and a difference of .006 for the exit Mach number, M_2 .

Variations could not be detected in the blade pressure readings except on the trailing edge.

All measurements, for a complete test (wall static, blade static and probe pressures distribution), usually are carried out in one run (in about 40-60 seconds) or in two runs with no tangible difference on the results.

The velocity distribution on the blade surfaces has been measured for different outlet flow conditions, from high subsonic to supersonic Mach numbers, less than the limited loading value; in Fig. 3 are shown some of these results, while in Fig: 4 'local' Mach number distribution is compared to 'critical' local Mach number, for a typical test.

Traversing probe measurements are also carried out for different outlet flow conditions but, later on, will be presented results relative to a typical flow condition.

Downstream Pts, Pss, DPlr, traverse pressures and upstream total pressure measurements, over a typical 45 seconds run time, are given in Fig. 5

The equivalent data reduced by probe calibration curves are shown in Fig.6.

The performance of the cascade has been calculated both using mass and area averaged values at the outlet cross section and the flow parameters at infinity down stream.

The results obtained on 2 pitches are shown respectively, in Fig. 7, for the exit Mach number, in Fig. 8, for outlet flow angle, and in Fig. 9, for the loss coefficient.

The difference between the average of the Mach numbers calculated on the two pitches is about 0.02.

This confirms the good periodicity of the exit flow.

The performance parameters calculated with the three methods show no appreciable difference (Table 1.).

4. Conclusions.

The results of research on a linear cascade of nozzles in a supersonic wind tunnel at D.I.M.E.CA. have been presented in this paper.

Under conditions where the scan time of the scanivalves was varied between 50ms and 1s and where the speed traversing probe was varied between 1 - 2.5mm/s, notwithstanding the brief time available for each test, the system of measurement for the analysis of the flow with a downstream aerodynamic probe has been consistently reliable and the results demonstrate a good accuracy.

The test rig and methodology are suitable, therefore, within a contained framework of costs, to undertake fundamental research on aspects of interest in the design of turbomachinery, such as performance under various operating conditions, investigation of trailing edge shock phenomena and the influence of coolant ejection effects.

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	M1 = .1766		$\beta_1 = 90^\circ$	
	M2	β_2	ξ_2	THL
Area averaged	1.2164	73.23	.0425	.0550
Mass averaged	1.2148	72.98	.0410	.0530
Infinity downstr.	1.2093	73.22	.0492	.0636

Table 1

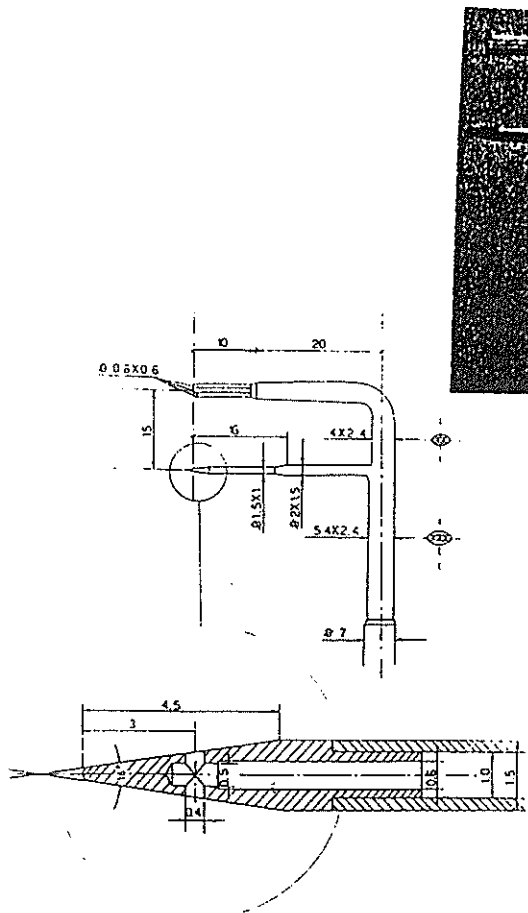
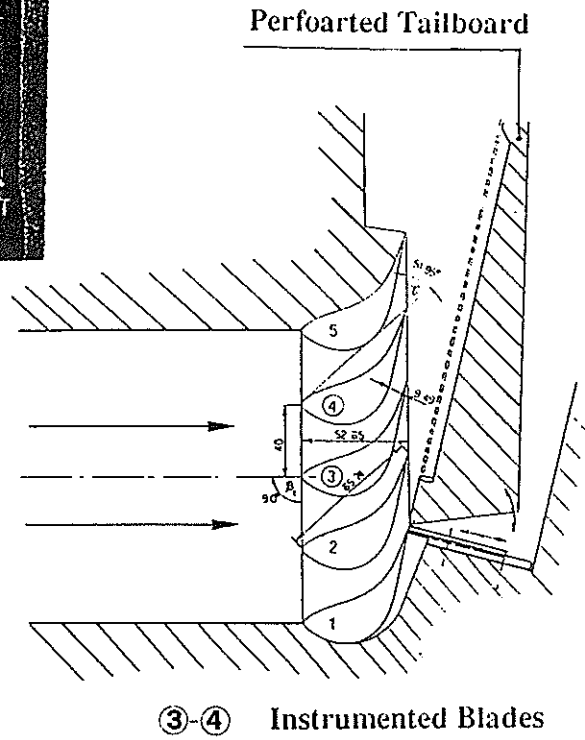
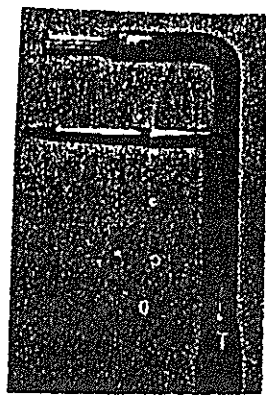


Fig. 1



③-④ Instrumented Blades

Fig. 2

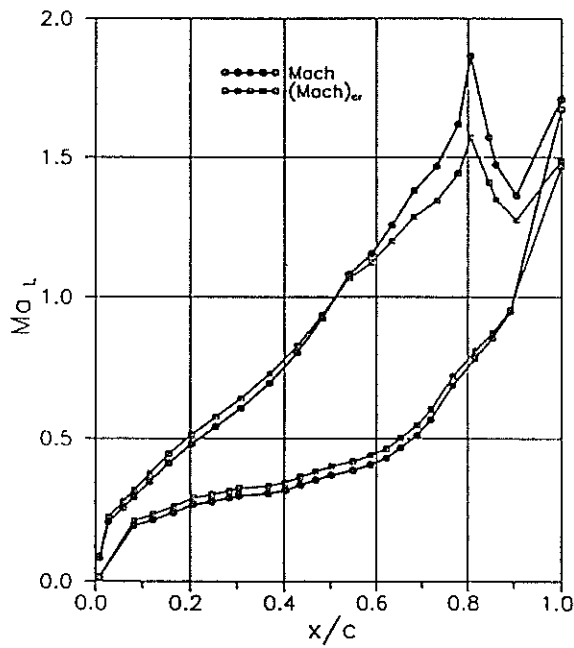


Fig. 4

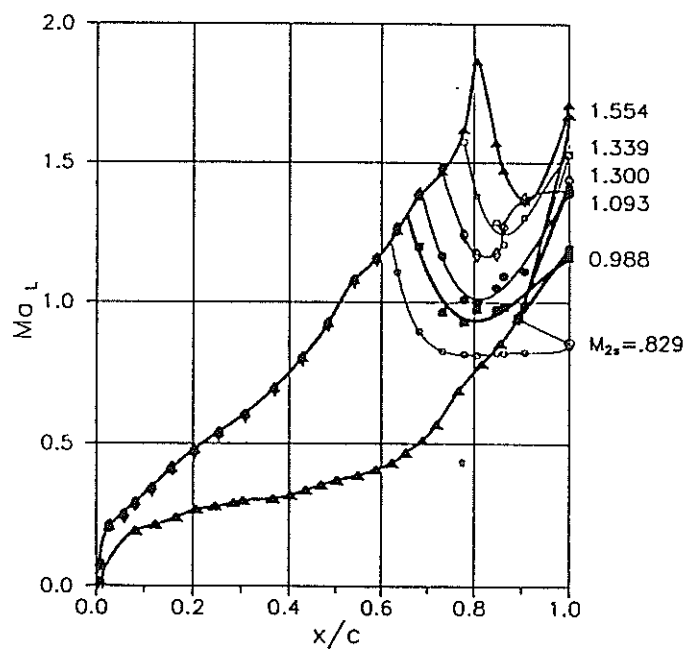


Fig. 3

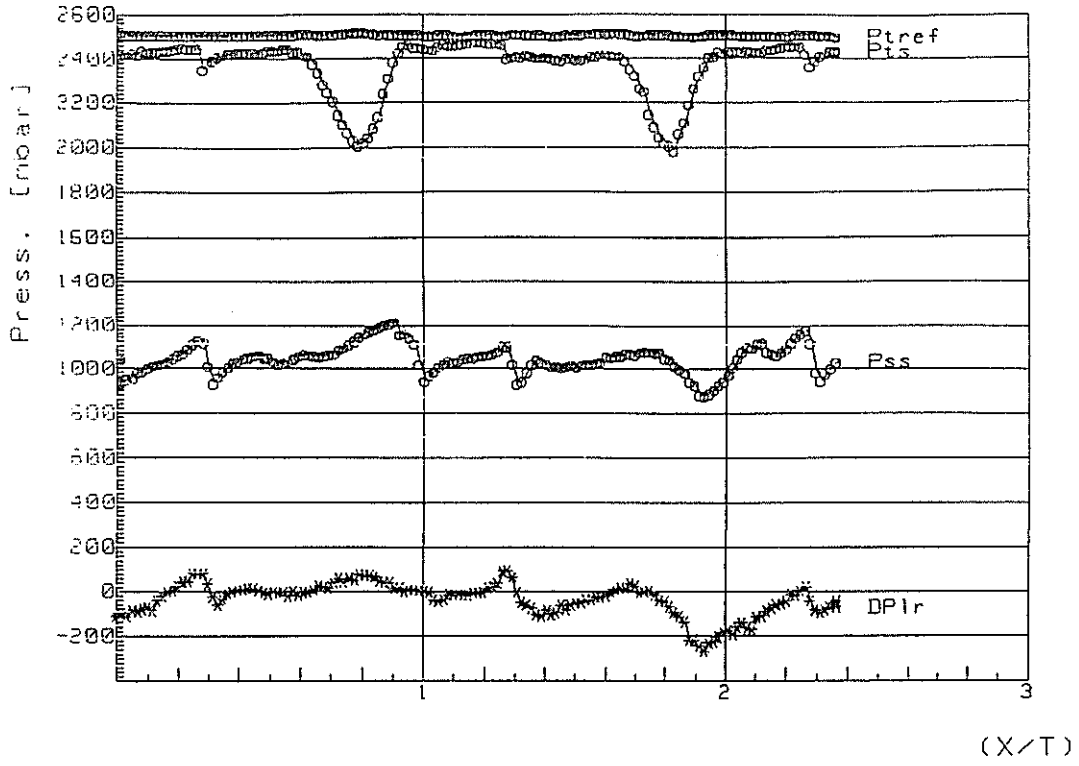


Fig. 5.

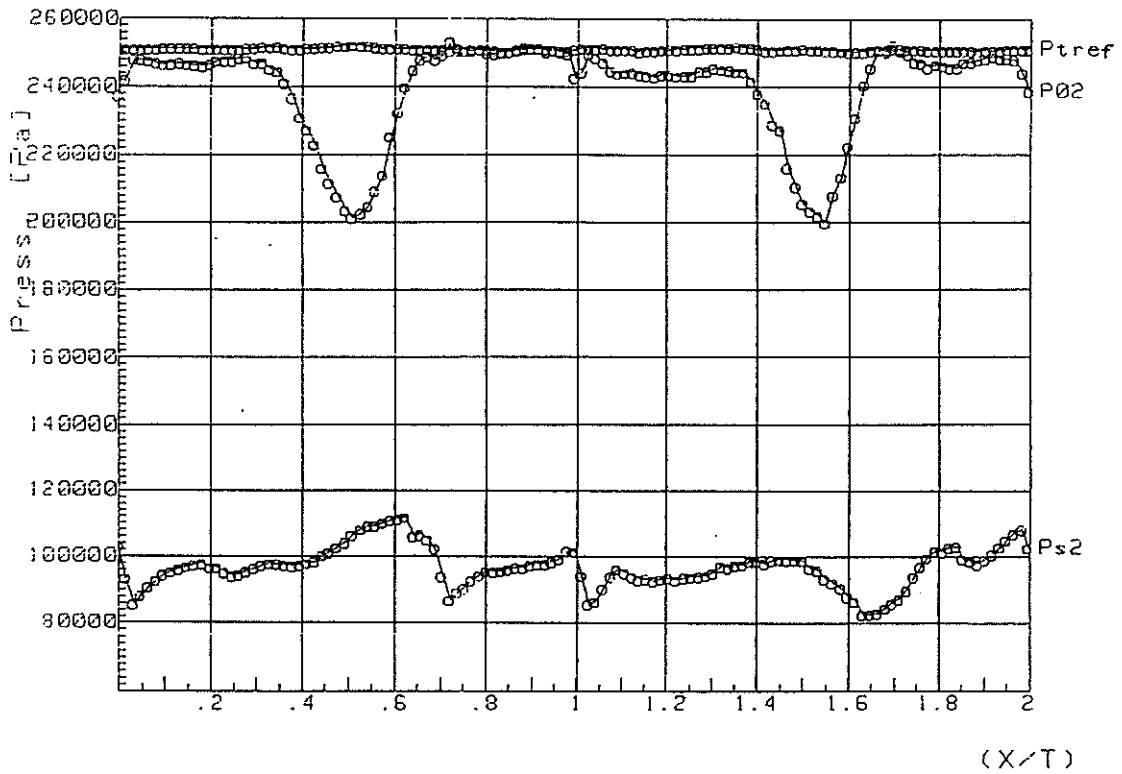


Fig. 6

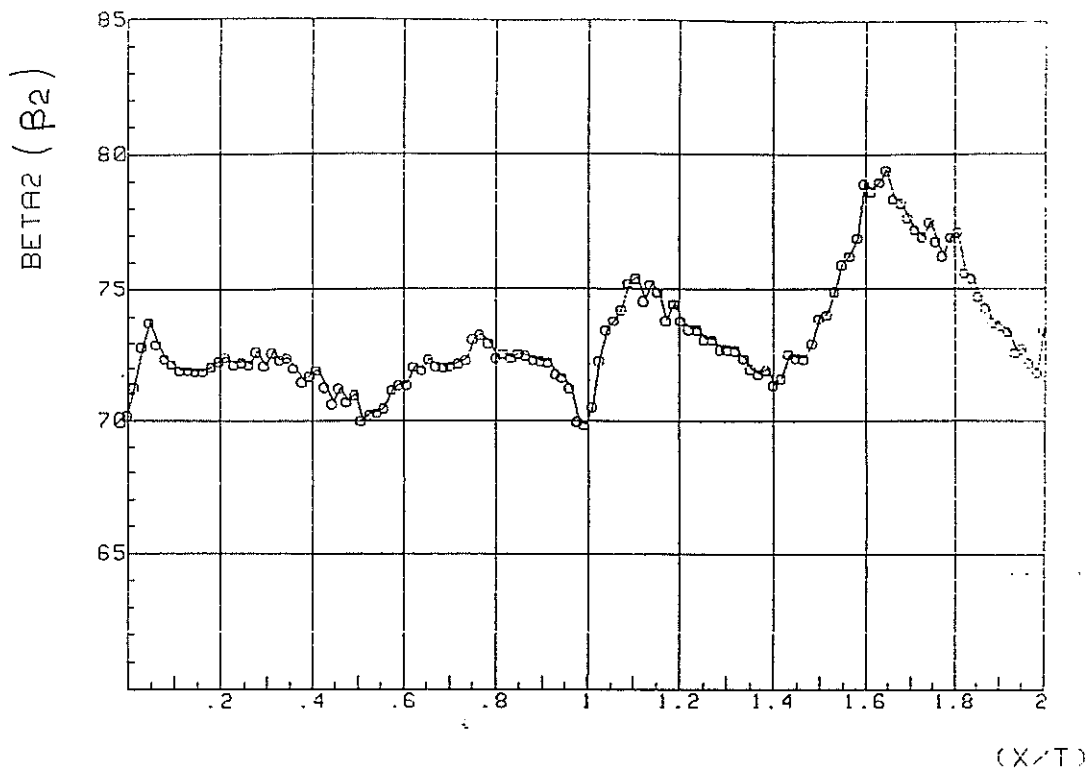


Fig. 7

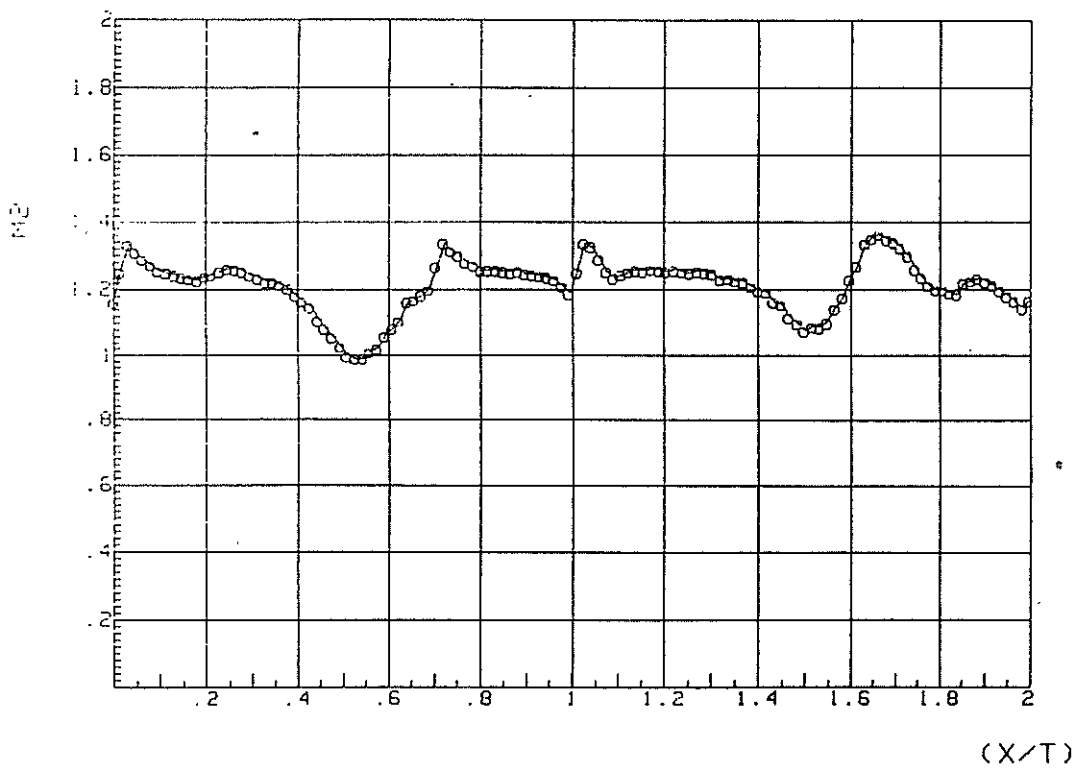


Fig. 8

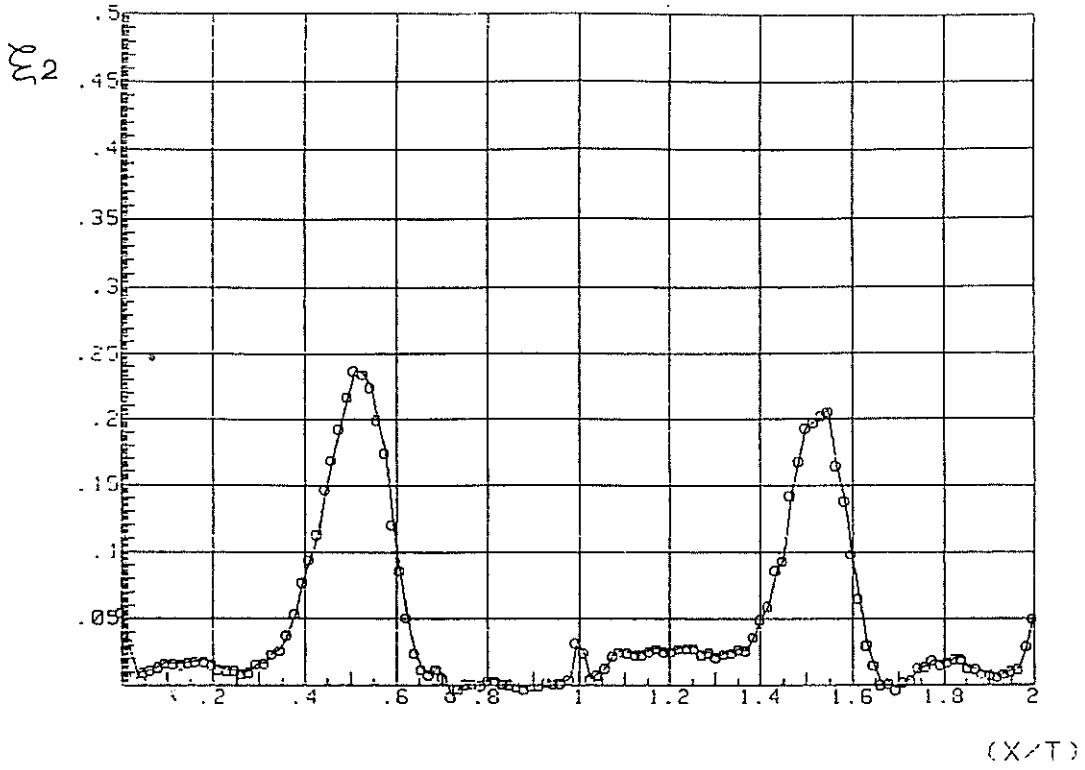


Fig. 9